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RESEARCH & EDUCATION IN AIRCRAFT DESIGN  
WARSAW, POLAND | 6-8 NOVEMBER 2024



## Input Data Determination for Assessment of Flutter Resistance of Small Aircraft

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### Abstract

In recent years, the category of small aircraft, particularly those with a maximum take-off weight of up to 600 kg, has seen significant growth in Europe. Previously, aeroelastic (AE) evaluations were not required for ultralight aircraft up to 450 kg, except for specific designs or those with higher cruise speeds. However, aeroelastic assessments are now mandated for UL2, CS-VLA, and CS-LSA type certifications. This paper focuses on the determination of input data required for flutter resistance analyses of small aircraft, emphasizing the importance of stiffness characteristics and ground vibration testing (GVT). Aeroelastic analyses are conducted using finite element models (FEM), and results are validated against GVT outcomes. The paper presents findings from seven aircraft tested at the Aviation Institute, with a detailed evaluation of the effects of input data scatter on the determination of critical flutter speeds. A particular focus is placed on the influence of composite materials and their uncertainty in structural properties, which impacts the reliability of aeroelastic analyses.

**Keywords:** Aeroelasticity, FEM, Ground vibration test, P-K method

### 1. Introduction

The category of small aircraft up to 600 kg maximum take-off weight has developed significantly in recent years, particularly in Europe. Previously, ultralight aircraft up to 450 kg did not require an aeroelastic assessment (AE), except for unconventional designs or aircraft with cruise speeds above 200 km/h. However, these assessments are now mandatory for UL2, CS-VLA and CS-LSA type certificates. This requirement is mainly due to the rapid increase in airspeed of aircraft in this category, driven by new design approaches, the use of composite materials and increasing engine power.

Aeroelastic analysis deals with the interaction of three types of forces, i.e. elastic, inertial and aerodynamic forces. This interaction leads to a number of aeroelastic phenomena under certain conditions, flutter being one of the most dangerous. The methods used to analyse the resistance of an aircraft to flutter and other aeroelastic phenomena have been developed and established progressively during the development of aviation since its beginning. The first methods were limited to very approximate solutions where the aerodynamic action was considered quasi-stationary, e.g., the use of Galerkin solutions [1] of the differential equations of motion describing the aeroelastic system at its stability limit as described in [2]. A more accurate determination of the dynamic aeroelastic stability limit required the study and establishment of methods for determining the aerodynamic forces and moments on the oscillating airfoils and wings. These efforts led to the establishment of more accurate methods for flutter calculation. One of the first methods in practical use is referred to as the "V-g" method [3]. This method works with the concept of structural damping, which allowed the graphical representation of the flutter solution in a diagram of the dependence of (artificial) damping on airspeed.

With the development of computer technology and numerical methods for solving structural and aeroelastic problems, methods for modal aeroelastic analysis such as the k-method and the p-k-method have been developed [4]. These methods are still widely used today and are also

implemented in computational tools. Over time, the standard aeroelastic tool for strength analysis has become the product MSC.NAstran, which of course includes aeroelastic analysis with evaluation in the form of "v-g" diagrams, both for the subsonic and supersonic regime.

In general, it could be assumed that there are no complications in this area and that classical methods will guarantee a reliable calculation of the critical flutter rate. However, it turns out that the main obstacle in the analyses is the lack of the main stiffness characteristics to reliably determine the required critical velocities. Especially for composite aircraft, the scatter of material properties is significant. A number of researchers have addressed this issue by approaching the problem by introducing uncertainties (aerodynamic, mass and stiffness) into the equations describing flutter [5], [6], [7].

Based on the results of aeroelastic analyses of a number of small sport aircraft, this paper presents the results and experience of the Institute of Aeronautics of the Brno University of Technology in evaluating the effect of input data scatter in terms of stiffness on the resulting determination of the critical flutter speed.

## 2. Certification requirements

The regulations governing the design of the aircraft category addressed in this paper are CS-VLA and CS-LSA. The requirements for certification of resistance to flutter and other aeroelastic phenomena are generally based on CS-23.

Under CS-VLA, which applies to very light airplanes with a maximum take-off mass of up to 750 kg, it is necessary to demonstrate that the airplane is free from flutter, control reversal and divergence under all operating conditions within the V-n envelope and at all speeds up to the speed specified for the selected certification method. These methods include the application of a rational analysis, which must show that aeroelastic phenomena will not occur at all speeds up to  $1,2v_D$ , and aeroelastic flight tests, which must meet the prescribed criteria. In addition, tolerances must be established for quantities affecting flutter such as velocities, damping, mass balance and control system stiffness. If the  $V_D$  speed is higher than 259 km/h, the natural frequencies of the main structural units must also be determined by vibration tests or other approved methods.

The CS-LSA regulation, which applies to light sport airplanes with a maximum take-off mass of 650 kg, requires ground vibration tests followed by analysis of vibration modes and frequencies and potential flutter events for airplanes with a VNE exceeding 200 km/h.

Both regulations allow the demonstration of aeroelastic performance of specified conventional airplanes by meeting the stiffness and mass balance criteria specified in Airframe and Equipment Engineering Report No 45. However, this option is only applicable to conventional airplanes that meet the design criteria specified in the regulations.

## 3. Ground vibration tests

This section outlines the procedure for experimentally obtaining the dynamic structural properties of the aircraft. It covers the general test requirements, including aircraft suspension, sensor and exciter placement, and data evaluation. The final part of the section provides a summary of the results from ground vibration tests performed on several aircraft at the Institute of Aerospace Engineering, Brno University of Technology.

### 3.1 Overview of ground vibration testing of aircraft

The basis for the analysis of aircraft resistance to flutter and other aeroelastic phenomena is the identification of the dynamic behavior of the aircraft structure, including natural frequencies, mode shapes and damping ratios. This analysis is typically performed through ground vibration testing of the airplane in critical configurations, with emphasis on mass configuration – light and heavy and the control system state – free and fixed control stick and pedals.

The ground vibration test (GVT) is conducted in accordance with Advisory Circular AC23-629-1b [8], which not only covers the GVT but also provides general information for flutter proofing.

During the measurement, the aircraft is suspended by a bungee-based soft support with a natural frequency less than half the lowest natural frequency of the aircraft structure. The aircraft is normally suspended by its landing gear legs as it is shown in Figure 1. This suspension provides free-free

conditions similar to those experienced by the aircraft in flight.



Figure 1 - The aircraft suspension during the ground vibration testing

The excitation of vibrations during the GVT is done by electrodynamic shakers located in appropriate positions. In our case it is usually at the wings tip, elevator tips and rudder tip locations. During the identification of control surface dynamic properties, the shakers excitation is brought to the surfaces at their trailing edge. The force is transmitted to the structure via a stinger rod, eliminating lateral inputs. The excitation force is measured by a force transducer positioned between the stinger rod and the aircraft structure. An auxiliary part was attached to the sensor.

The response is measured using the accelerometers located at the aircraft in such manner that it is possible to identify the most important mode shape of the airframe vibrations. The typical layout of responses and excitation location is illustrated in Figure 2 and Figure 3.

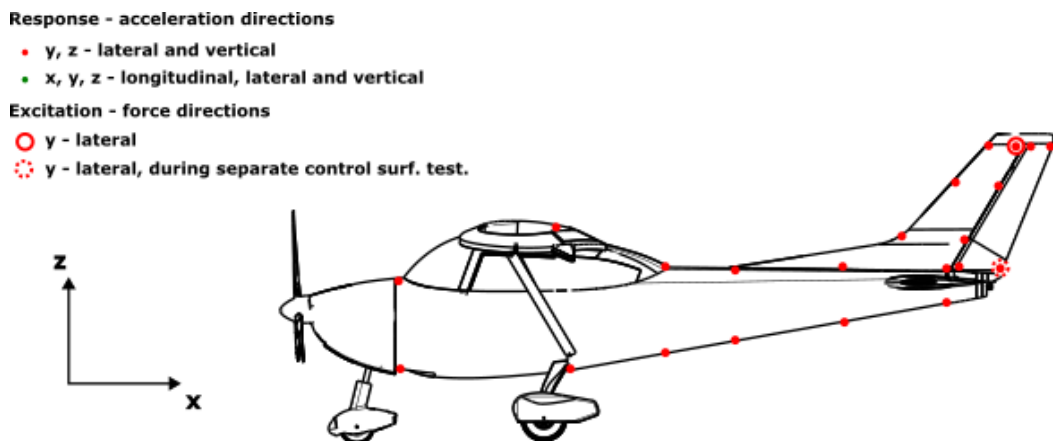


Figure 2 – Example of the layout of responses and excitation during GVT, side view

## Input Data Determination for Assessment of Flutter Resistance of Small Aircraft

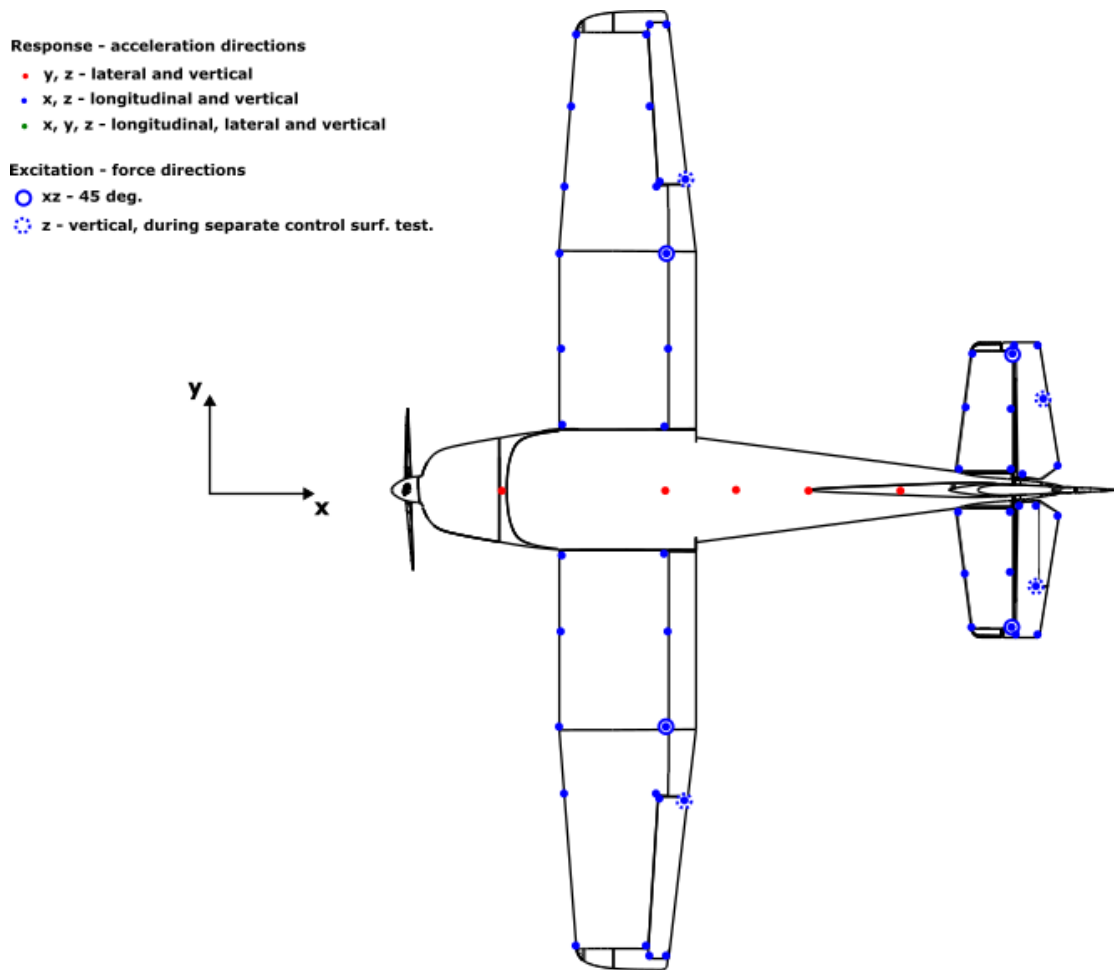


Figure 3 - Example of the layout of responses and excitation during GVT, top view

Vibrations are excited using a stepped-sine signal within the frequency range of 5 to 105 Hz, with a frequency step size of 0.25 Hz and a settling time of 10 periods per step. During the initial run, the exciters operate in phase. In the second run, the phase of one exciter is shifted by 180 degrees, creating an out-of-phase excitation pattern.

Due to the limited number of sensors available, the roving response method is employed. The measurement process is segmented into multiple stages, with accelerometers being relocated after each stage to acquire a complete response profile.

The software utilized for pre-processing, measurement, and post-processing tasks is BK Connect [9]. Temporal data from the accelerometers and force transducers is processed in real-time using Fast Fourier Transformation (FFT) to compute the frequency response function. This function is subsequently analysed using specific criteria, typically the Rational Fraction Polynomial-Z (RFP-Z) method, resulting in the generation of a stabilization diagram (see Figure 4 for an example). From this diagram, the natural frequencies, damping ratios, and corresponding mode shapes of the aircraft structure are extracted and identified.

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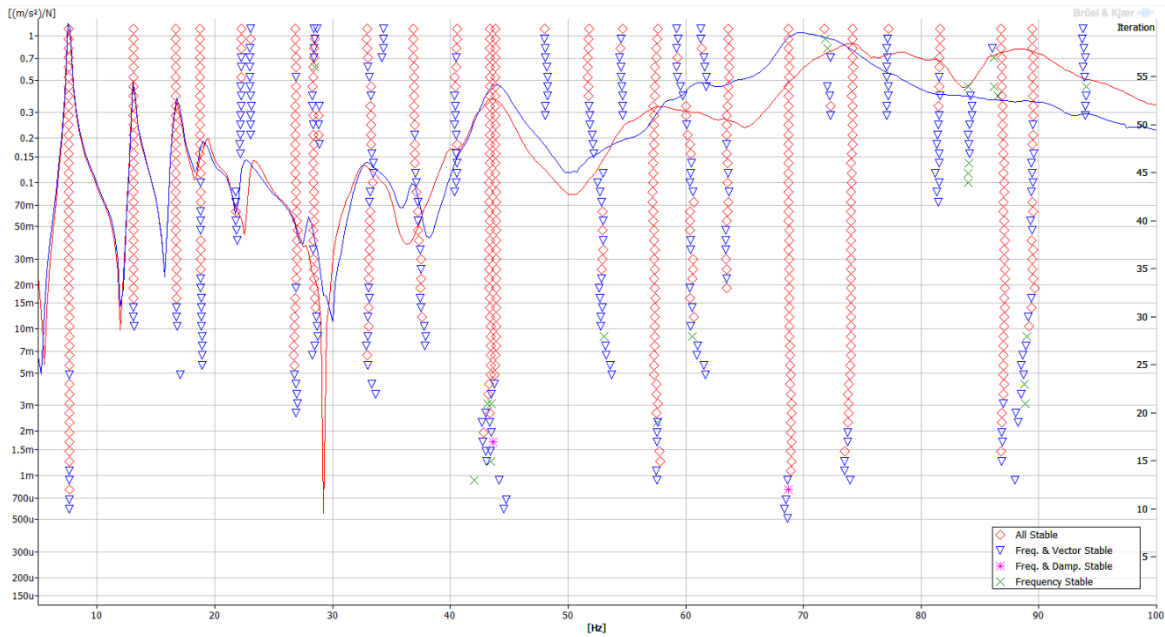


Figure 4 - Example of stabilization diagram

### 3.2 Results of Ground vibration tests

Flutter resistance and other aeroelastic phenomena have been successfully tested at the Institute of Aerospace Engineering for a total of seven aircraft, each with a maximum takeoff weight of 600 kg. Three of the aircraft had airframes made from aluminum alloy, while the remaining were constructed from composite materials.

The outcome of the ground resonance test for each aircraft includes the determination of natural frequencies, vibration modes, and the damping ratio of the airframe structure and its control surfaces. The natural frequencies for selected vibration modes of the airframes are listed in Table 1, where the aircraft are categorized by type of material and wing configuration.

Table 1 - Natural frequencies of selected mode shapes of several aircraft measured at IAE, frequencies are in Hz

	Composite					Metal	
	low wing	low wing	low wing	low wing	high wing	low wing	high wing
wing 1st symmetric bending	8.0	8.3	10.0	7.6	11.6	12.3	14.8
wing 1st antisymmetric bending	18.4	17.5	18.4	16.7	19.9	32.8	30.5
tail roll	12.2	11.8	14.4	13.2	16.4	15.7	15.2
fuselage 1st vertical bending	18.7	12.5	28.5	18.7	25.1	-	20.1
fuselage 1st lateral bending	22.8	21.7	18.2	21.8	19.3	21.7	17.6
stabilizer 1st symmetric bending	30.2	29.4	28.5	27.6	25.1	38.7	41.5
fin 1st bending	28.8	21.7	33.6	29.5	24.2	28.5	45.4

The identified natural frequencies of control surfaces rotation for the same aircraft are listed in Table 2 for the free and fixed control stick and pedals.

Table 2 - Natural frequencies of control surface rotation with free and fixed control stick and pedals, frequencies are in Hz

	Composite					Metal	
	low wing	low wing	low wing	low wing	high wing	low wing	high wing
<b>Free control</b>							
Antisymmetric aileron rotation	37.4	21.2	16.2	40.5	17.4	16.7	8.8
Symmetric elevator rotation	37.6	8.6	18.8	31.3	14.9	14.8	17.0
Rudder rotation	16.3	25.5	20.0	17.2	10.1	13.1	18.2
<b>Fixed control</b>							
Antisymmetric aileron rotation	39.9	30.8	31.5	41.4	17.3	19.2	19.2
Symmetric elevator rotation	37.6	19.8	19.2	31.4	16.7	17.2	16.9
Rudder rotation	16.6	24.9	22.3	19.6	10.1	13.3	18.0

#### 4. Finite element model for flutter analysis

This section provides a detailed overview of the preparation of the elastic and mass finite element models, the validation of the computational modal analysis against results of the ground vibration test, and the development of the aeroelastic model. The concluding part presents the results of the flutter analysis. Additionally, a discussion on the balancing of control surfaces is included, highlighting its critical role in preventing flutter below the  $1.2v_D$  limit.

##### 4.1 The elastic finite elements model

The primary input for aeroelastic analysis is the elastic model of the complete aircraft. Typically, this model is built using 1D BEAM elements with properties defined by a PBEAM card to represent the entire structure, see Table 1. These elements are positioned along the elastic axes of the fixed components, such as the wing, stabilizer, fin and fuselage, and along the elastic axes of the control surfaces, such as the ailerons, elevators, rudder and trim tabs.

The elastic behaviour of these elements is characterised by four key properties: the cross-sectional area  $A$ , the two perpendicular second moments of the area  $I_1$  (usually the bending of the wing) and  $I_2$  (the in-plane bending of the wing) and the torsional constant  $J$  and the material properties. These parameters are derived from the geometry and material properties of the respective structural sections. The exact determination of these properties is complex and often subject to considerable uncertainty. Therefore, it is not possible to define the exact position of the elastic axis, the cross-sectional area, the bending stiffness in two directions and the torsional stiffness for each cross-section.

Table 3 - BEAM element properties description

PBEAM	PID	MID	A	I1	I2	J	
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where:

*PID* is property identification number.

*MID* Material identification number

*A* Area of the beam cross section

*I1* Area moment of inertia for bending in plane 1 about the neutral axis.

*I2* Area moment of inertia for bending in plane 2 about the neutral axis.

*J* Torsional stiffness parameter

The traditional division of aircraft structures into beam elements for bending and torsion boxes for torsional loads is not sufficient to accurately determine stiffness properties. In reality, the entire structure contributes to both bending and torsional stiffness, a phenomenon that becomes even more significant when composites are used. Computer aided design (CAD) systems can assist in the

determination of bending moments of inertia by analyzing the cross-sectional geometry of the structure (Figure 5). These systems can provide more accurate calculations for both bending and torsional stiffness, considering the contributions of the entire structural configuration.

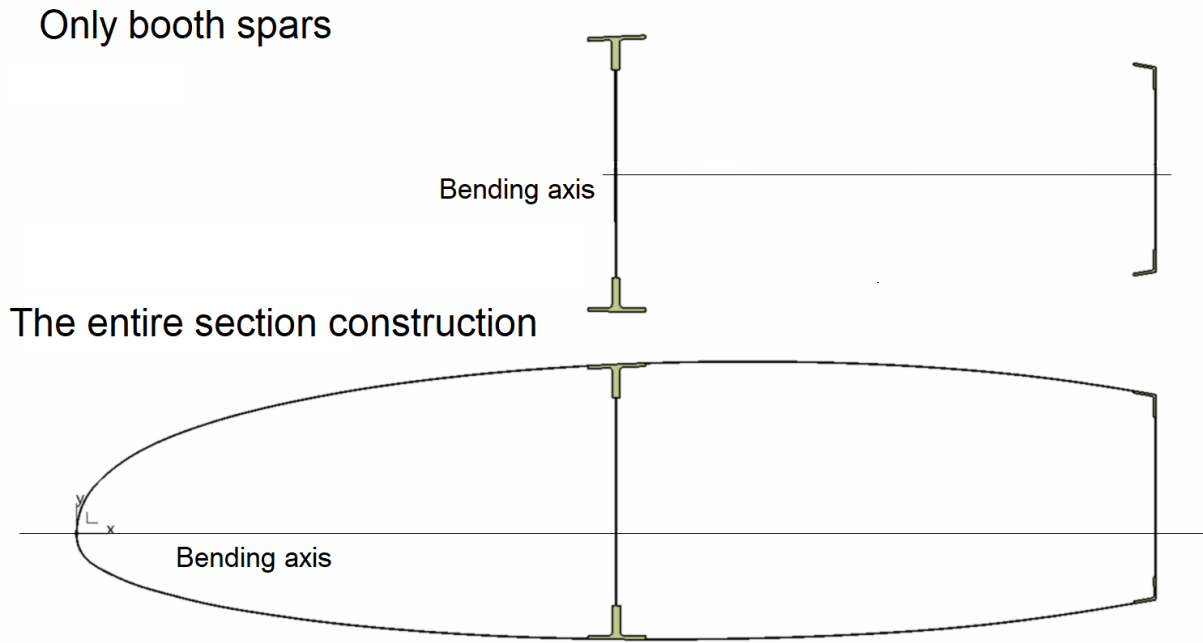


Figure 5 - Determination of second moments of area using CAD

The difference between the second moments of inertia  $I_1$  (wing bending) calculated by dividing the structure into bending resistant components and those calculated for the entire structure as a whole is considerable as it is shown in example in the Table 4.

Table 4 - Comparison of Second moments of area

	Second moments of area of only booth spars	Second moments of area of entire construction of the section	Delta
$I_1$ [mm <sup>4</sup> ]	12 293 998.0	21 275 900.0	42.22%

However, while CAD systems can accurately determine bending stiffness, they cannot calculate torsional stiffness. This requires the use of analytical methods.

The formula for torsional constant  $J$  for torsion box (single cell) is:

$$J = \frac{4A^2}{\oint \frac{ds}{t}} \quad (1)$$

where  $A$  is area of cell,  $t_i$  is thickness of skin or spar web and  $s_i$  is length of skin or spar web.

This equation is valid for a single cell. However, many aircraft structures consist of several closed sections. A simplified formula can also be derived for a double cell structure.

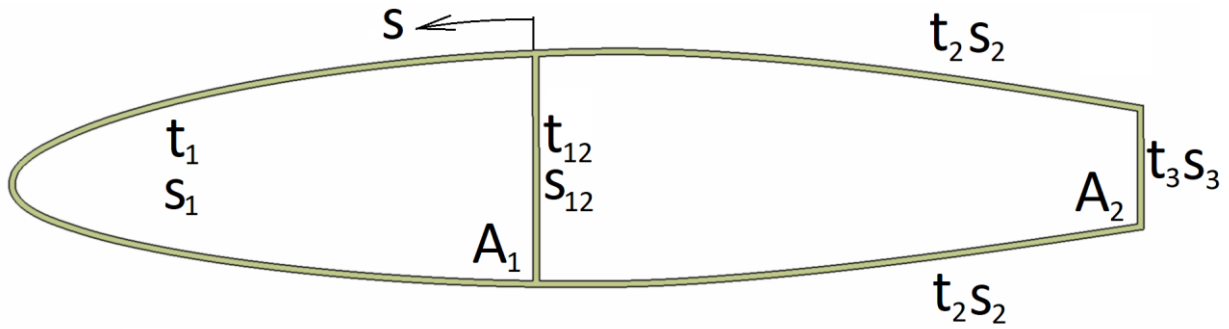


Figure 6 – Torsion constant for double cell

$$J = 4 \left[ \frac{a_{20}A_1^2 + a_{12}(A_1 + A_2)^2 + a_{01}A_2^2}{a_{01}a_{12} + a_{12}a_{20} + a_{20}a_{01}} \right] \quad (2)$$

$$\text{where } a_{ij} = \oint \frac{ds}{t} \quad (3)$$

where  $A_i$  is area of cell,  $t_i$  is thickness of skin or spar web,  $s_i$  is length of skin or spar web.

A comparison of the resulting second moments of inertia for single cell and double cell configurations is shown in the Table 5.

Table 5 - Comparison of torsional constants

	Singel cell	Double cell	Delta
J [mm <sup>4</sup> ]	40 557 434.91	40 092 559.11	1.15 %

## 4.2 The mass model

The mass model for aeroelastic analysis of small aircraft is developed based on a detailed mass analysis, the geometric model, and the actual weights of individual aircraft components provided by the manufacturer. For most components, both the precise weight and the center of gravity can be determined in this manner. In aeroelastic analysis, this data is incorporated into the FEM mass model, typically using 0D CONM2 elements (with input data shown in Table 6), which allow the specification of both mass and moments of inertia. However, accurately determining the moments of inertia for individual components is often complex.

To address this challenge, the mass FEM model can approximate moments of inertia by dividing the individual components into a greater number of CONM elements. This method effectively distributes the mass and replaces the need for directly inputting moments of inertia. The FEM model can then manage both the mass of the components and the position of their centers of gravity, such as for the wing or control surfaces like the rudder. Figure 7 illustrates a typical FEM model, where 1D elements represent elastic axes and 0D elements represent mass distribution.

Table 6 - Mass element properties description

CONM2	EID	G	CID	M	X1	X2	X3
	I11	I21	I22	I31	I32	I33	

where:

*EID* Element identification number.

*G* Grid point identification number

*CID* Coordinate system identification number

*M* Mass value

*X1,X2,X3* Offset distances from the grid point to the center of gravity

*Iij* Mass moments of inertia measured at the mass center of gravity



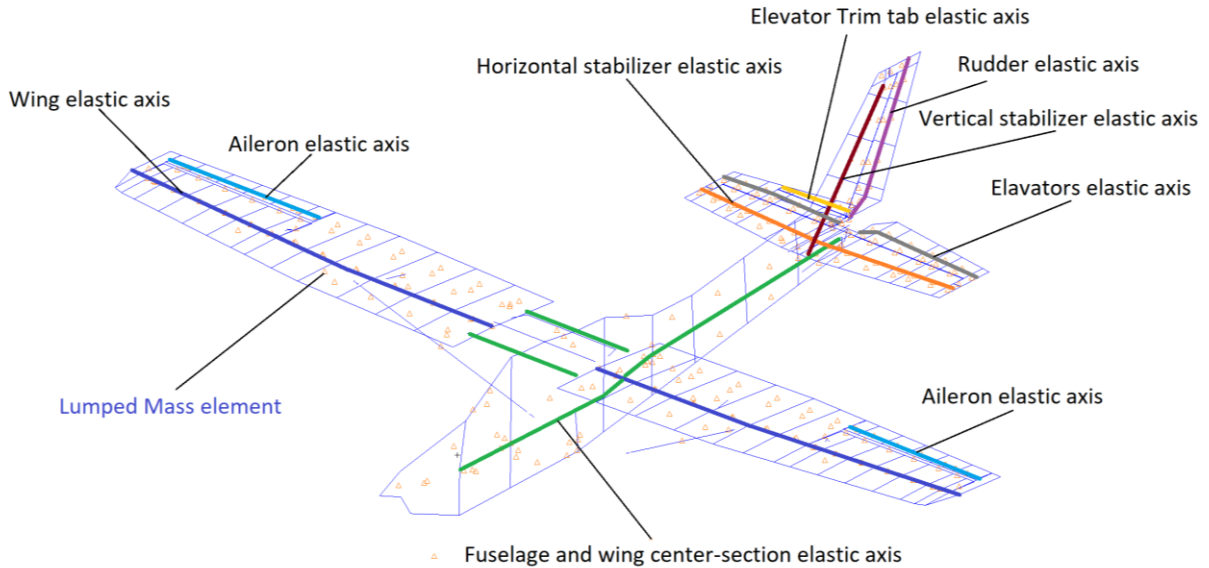


Figure 7 - Aeroelastic FEM model

### 4.3 Finite elements model of control

The FEM model of the control linkage within the elastic FEM model can largely be replaced by CBUSH spring elements, with their properties defined through PBUSH cards (Table 7). It is not necessary to model the entire control linkage; most of the linkage can be substituted with a spring element of appropriate stiffness and damping. However, this approach cannot be applied to the aileron control linkage, where it is essential to maintain the connection between the left and right ailerons, as shown in Figure 8. A spring can be introduced into the linkage only after establishing the connection between the left and right ailerons.

Table 7 - BUSH element properties description

PBUSH	PID	K	Ki				
		GE	GEi				

where:

- PID* Property identification number
- K* Flag indicating that the next 1 to 6 fields are stiffness values in the element coordinate system
- GE* Flag indicating that the next fields, 1 through 6 are structural damping constants
- Gei* Nominal structural damping constant in directions 1 through 6

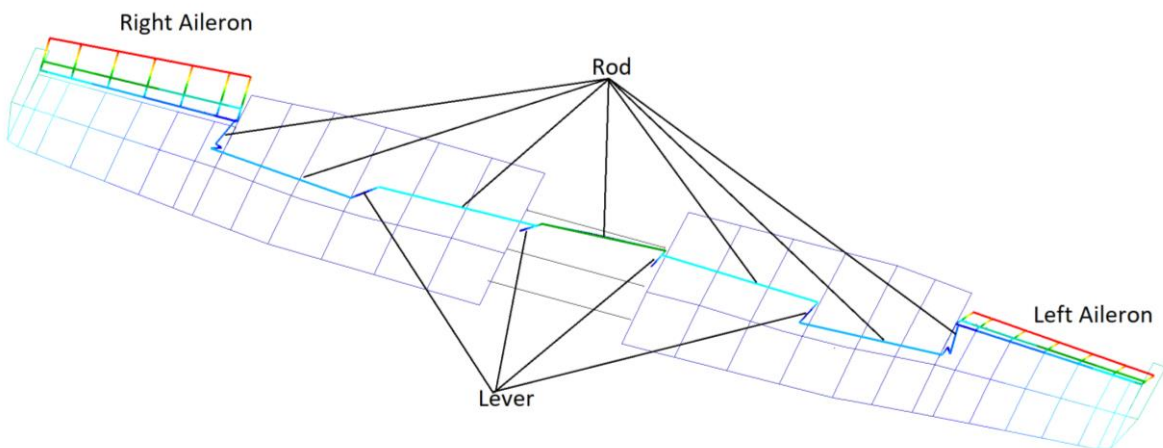


Figure 8 - Aileron control linkage (connection between left and right)

The stiffness of the springs used is determined for both locked and free control cases. These values are calibrated so that the oscillations of the control surfaces (their rotation around hinge points) match the results of the ground resonance test.

#### 4.4 Verification of the FE model

From the previous chapters, it is evident that the initial FEM model for assessing aeroelastic resistance is significantly inaccurate and requires comparison and calibration against the results of the ground vibration test. This can be achieved through modal analysis. The outcomes of these analyses include the natural frequencies and mode shapes of the FEM model. These mode shapes and frequencies are then compared with the results of the ground resonance test. The comparison and subsequent tuning of the FEM model must be conducted in several steps:

- Initial rough tuning of the entire model
- Tuning the stiffness of the springs that substitute for the control linkage stiffness (both locked and free controls)
- Final tuning of the FEM model stiffness for locked controls (light and heavy configurations)

This is achieved using modal analysis and SOL 200 [8] in MSC.Nastran, which allows for parametric optimizations. The objective for optimization is to minimize the deviation of FEM frequency modes from ground vibration test frequencies:

$$\min f = \sum_{i=1}^n X_i \left( \frac{F_i^{GVT} - F_i^{FEM}}{F_i^{GVT}} \right)^2 \quad (4)$$

where  $X_i$  is the weight for frequency  $f_i$  (typically 1, with higher values for low frequencies),  $F_i^{GVT}$  is the  $i$ -th natural frequency from GVT and  $F_i^{FEM}$  is the  $i$ -th natural frequency from the FE model.

The optimization parameters for each element of the elastic axis, which represents the whole aircraft, consist of two values of bending stiffness and one value of torsional stiffness.

A critical condition of this process is the correct pairing of the mode shapes from the finite element analysis with the shapes from the ground resonance test. The optimization is performed by minimizing the objective function (4). Optimization is then followed by re-determination of the FEM natural frequencies and comparison of the frequencies and shapes with the GVT.

A comparison of the FEM modal analysis results after cross-sectional characteristic optimization and the GVT results is shown in Table 8. The required difference between the individual frequencies is less than 10 %. If this requirement is not fulfilled, the second round of parametric optimization and comparison of results is necessary.

Table 8 - example comparison of GVT frequencies and FEM frequencies

Ground vibration test			FEM	
	f [Hz]	Mode shape	f [Hz]	difference
1	11.62	1 <sup>st</sup> symmetrical bending of the wing	11.68	-0.54 %
2	16.36	Horizontal bending of the fuselage; torsion of rear part of the fuselage; oscillation of rudder	16.71	-2.10 %
3	19.31	Horizontal bending of the fuselage; torsion of rear part of the fuselage; oscillation of rudder	18.39	4.78 %
4	19.85	2 <sup>nd</sup> anti-symmetrical bending of the wing with oscillation of the ailerons	20.44	-2.99 %
5	24.20	VS bending	24.85	-2.67 %
6	25.13	1 <sup>st</sup> HS bending; 2 <sup>nd</sup> vertical bending and torsion of the fuselage; symmetrical oscillation the elevator	25.80	-2.66 %
7	25.88	2 <sup>nd</sup> symmetrical bending-torsion of the wing with oscillation of the ailerons	25.80	0.29 %
8	31.23	1 <sup>st</sup> bending-torsion of the HS - asymmetrical/right half; anti-symmetrical oscillation elevator	32.44	-3.90 %
9	31.85	2 <sup>nd</sup> anti-symmetrical bending-torsion of the wing with oscillation of the ailerons	32.81	-3.01 %
...				
30	94.32	Bending-torsion of the wing	96.22	-2.01 %

$$difference = \frac{f_{GVT} - f_{FEM}}{f_{GVT}} 100 [\%] \quad (5)$$

Where  $f_{GVT}$  is frequencies from GVT and  $f_{FEM}$  is frequencies from FEM.

#### 4.5 Aeroelastic FE model

The aeroelastic analysis using MSC.Patran/MSC.Nastran relies on the finite element method supplemented with aerodynamic elements. The primary inputs include the structural stiffness represented by the elastic model, mass and moments of inertia represented by lumped masses, and aerodynamic excitation linked to the elastic model to provide structural excitation.

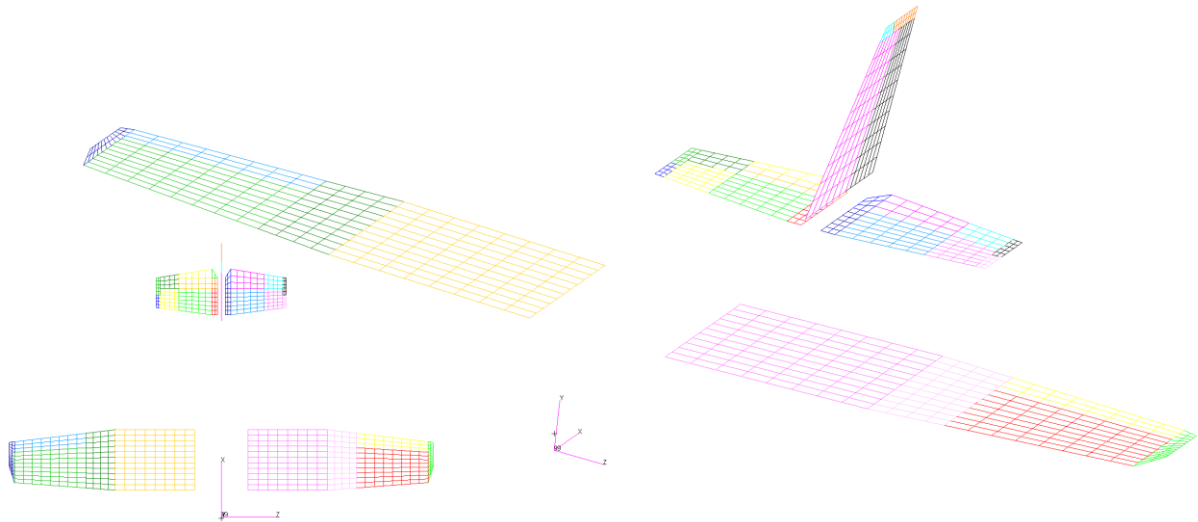


Figure 9 – Aerodynamic surface for aeroelastic analysis

#### Estimation of critical flutter speed – p-k method

The p-k method, which is widely used to calculate the critical flutter speed, is an iterative process. It is applied for each combination of airspeed, air density, and natural mode of the structure. During each iteration, the method solves for complex eigenvalues of the aeroelastic equation of motion (Eq. 6). These eigenvalues provide the damping value  $g$ , where zero damping indicates the aeroelastic stability limit of the mode and the corresponding airspeed is the critical flutter speed.

$$\left[ M_{hh} p^2 + \left( B_{hh} - \frac{1}{4} \rho \bar{c} V Q_{hh}^I / k \right) p + \left( K_{hh} - \frac{1}{2} \rho V^2 Q_{hh}^R \right) \right] \{u_h\} = 0, \quad (6)$$

where  $M_{hh}$  is the modal mass matrix,  $B_{hh}$  is the modal damping matrix,  $K_{hh}$  is the modal stiffness matrix,  $k = \omega c / (2V)$  is the reduced frequency,  $c$  is the reference depth,  $\omega$  is the circular frequency,  $V$  is the airspeed,  $\rho$  is the air density,  $Q_{hh}^R$  is the modal aerodynamic stiffness matrix,  $Q_{hh}^I$  is the modal aerodynamic damping matrix. These matrices are the real and imaginary components of the aerodynamic force matrix  $Q_{hh}$  and both are functions of the Mach number  $M$  and the reduced frequency  $k$ . The value of  $p$  in equation is the eigenvalue  $p = \omega(\gamma \pm i)$ , where  $\gamma$  is the transient decay rate, which is related to the structural damping coefficient  $g = 2\gamma$ .

Equation (6) is rewritten in state-space form for the p-k method as:

$$[A - pI] \{u_h\} = 0 \quad (7)$$

where  $[A]$  is real matrix

$$[A] = \begin{bmatrix} 0 & I \\ -M_{hh}^{-1} [K_{hh} - \frac{1}{2} \rho V^2 Q_{hh}^R] & -M_{hh}^{-1} [B_{hh} - \frac{1}{4} \rho \bar{c} V Q_{hh}^I / k] \end{bmatrix} \quad (8)$$

and vector  $\{u_h\}$  includes modal displacements and velocities.

The iterative process continues until the difference between the circular frequencies  $\omega$  obtained in consecutive iterations falls below a specified tolerance, ensuring convergence.

#### 4.6 Aeroelasticity analysis results

The results of the aeroelastic analysis are presented in the form of v-g and v-f diagrams (velocity-damping and velocity-frequency), which show, for each vibration mode of the aircraft structure, whether the oscillation is damped ( $g < 0$ ) and flutter does not occur within the given speed range, or undamped. Flutter occurs at the speed where damping reaches zero ( $g = 0$ ) for a specific vibration mode. The speeds  $V_{NE}$  (never exceed speed),  $V_D$  (design diving speed), and  $V_f$  (flutter critical airspeed) shown in the graph indicate whether flutter occurs at flight speeds greater than  $V_f = 1.2V_D$ , or below this threshold. Diagram in Figure 10 illustrates that for modes 7 and 8, flutter occurs at speeds lower than  $V_f$ , which is unacceptable from a flight safety perspective, and modifications are required. The diagram also shows that for mode 16, flutter occurs at a speed higher than  $V_f$ , providing a sufficient safety margin, so no further action is necessary. For the remaining modes, either flutter does not occur, or it occurs at speeds greater than 450 km/h, which is beyond the studied speed range.

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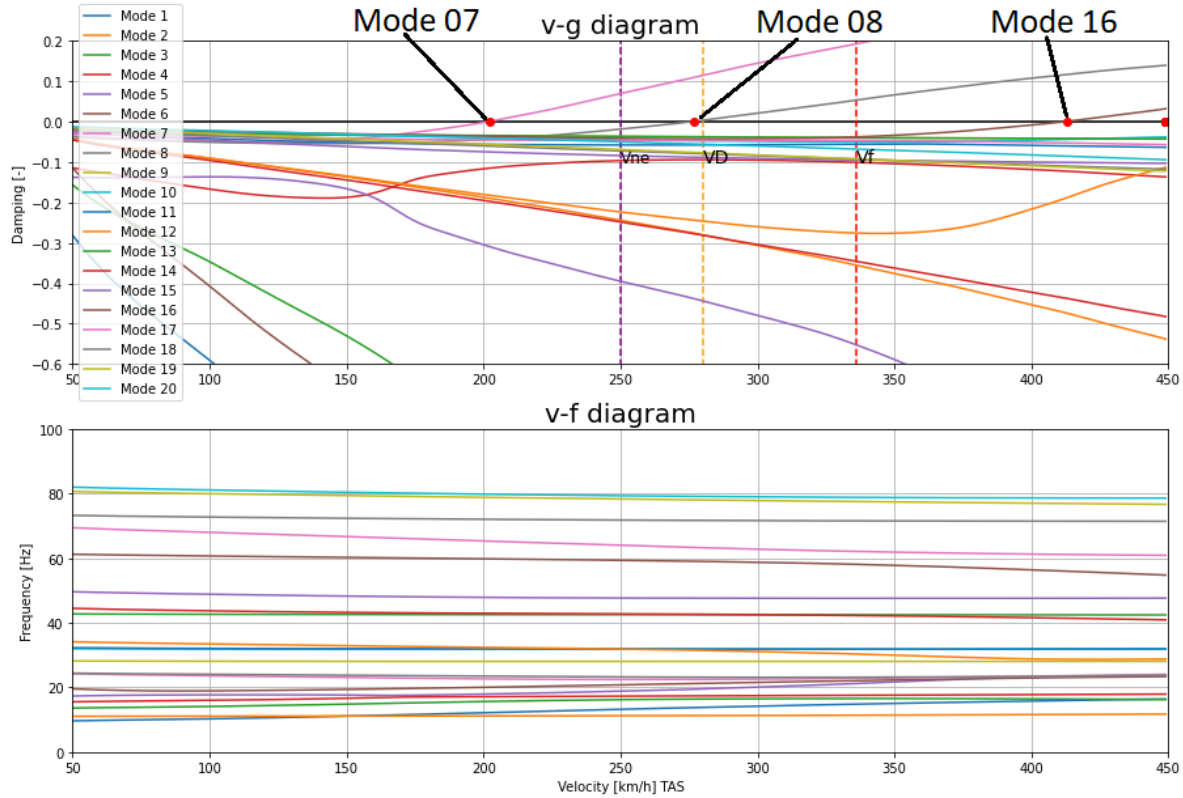


Figure 10 - Typical results of aeroelastic analysis

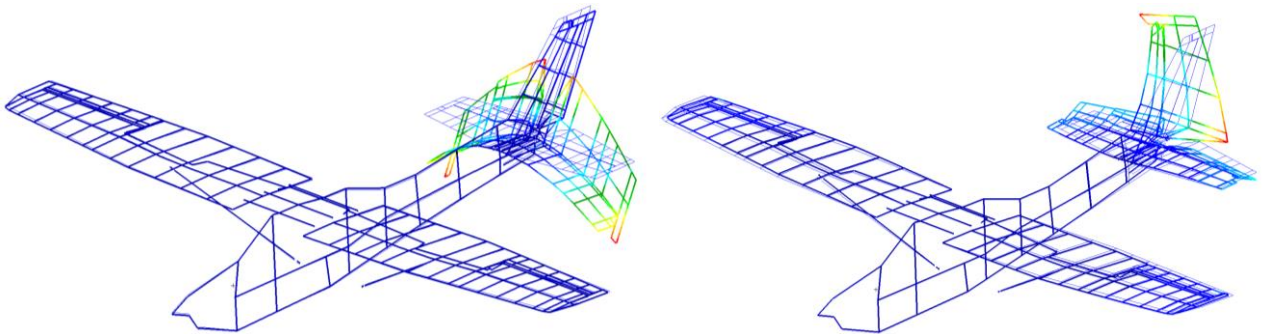


Figure 11 - Mode 07 (left) and Mode 08 (right) before the balanced elevators and Rudder

### 4.7 Effect of controls balance

Experience from analyses shows that flutter occurs mostly on unbalanced tail rudders and in the vast majority of cases it is a combination of aft fuselage oscillation and rudder or elevator oscillation. These phenomena can be avoided by at least partially balancing the rudders, thus shifting the flutter speed to higher speeds. This fact is illustrated in Figure 11 - Figure 13.

### Input Data Determination for Assessment of Flutter Resistance of Small Aircraft

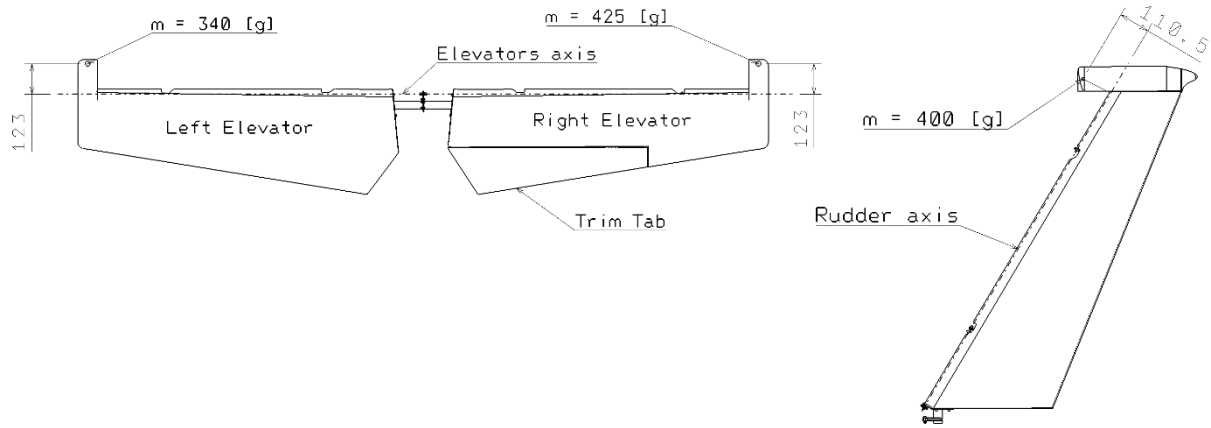


Figure 12 - Semi (minimal) balanced elevator and rudder

The following v-g diagram (Figure 13) shows the change in flutter speed for the variant without rudder balance and the variant with partial rudder balance (elevator and rudder) m. Adding masses in front of the rudder rotation axis (Figure 12) increased the critical flutter speed from 202 km/h to 339 km/h. Mode 07 even results in a complete damping of that mode.

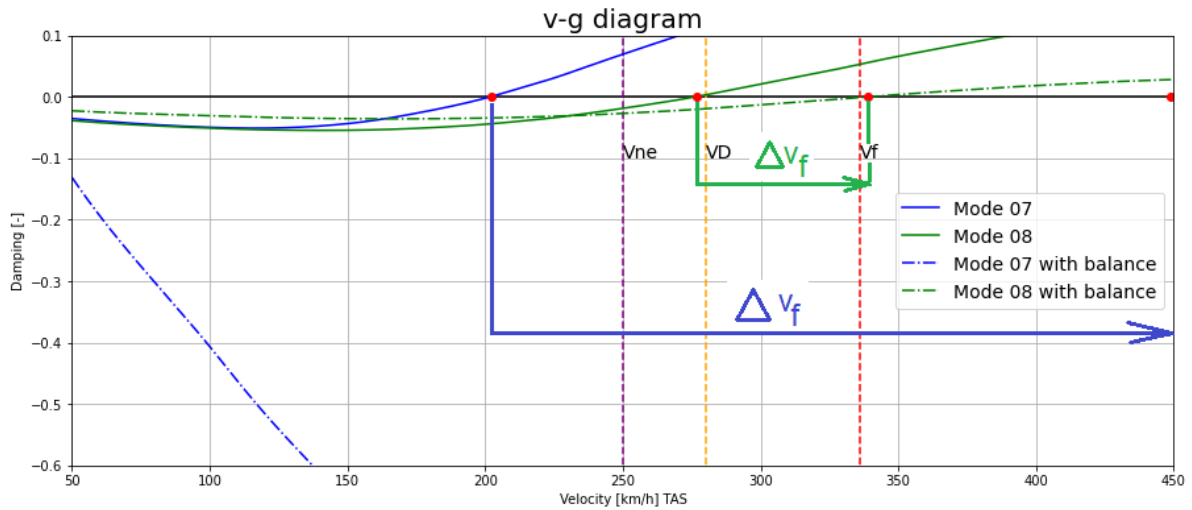


Figure 13 - v-g diagram for Mode 07 and 08 (original elevators and Rudder and elevators and Rudder with balance mass)

## 5. Conclusion

The results of aeroelastic analyses of small sport aircraft highlight the critical importance of accurate input data, particularly stiffness characteristics, for determining the flutter resistance of small aircraft. Ground vibration testing is essential to determine the natural frequencies, mode shapes and damping ratios of the aircraft structure.

Due to the uncertainties in the cross-sectional and material characteristics, especially for composite structures, entering the finite element model, it is necessary to perform parametric optimization of these cross-sectional characteristics to achieve agreement between the GVT results and the modal analyses of the aircraft model entering the flutter analysis. Under these conditions, the critical flutter speed can then be correctly determined.

Other findings from the flutter analyses performed (see chapter 4.7.) show that flutter occurs predominantly in unbalanced control surfaces such as rudders, and can be mitigated or eliminated completely by balancing, or at least partially balancing (adding mass across the axis of rotation of the rudder) these surfaces, effectively shifting the flutter rate to higher, safer values. These findings highlight the need for accurate control surface balancing and thorough testing to ensure aeroelastic safety in small aircraft designs.

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